Engineering Mechanics (3ME3-04)

DEPARTMENT

OF

MECHANICAL ENGINEERING

(Jaipur Engineering College and Research Centre, Jaipur)

UNIT: I

Statics of particles and rigid bodies: Fundamental laws of mechanics, Principle of transmissibility, System of forces, Resultant force, Resolution of force, Moment and Couples, Varignon's theorem, Resolution of a force into a force and a couple, Free body diagram, Equilibrium, Conditions for equilibrium, Lami's theorem.

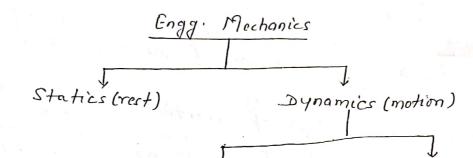
Plane trusses: Types of structures, Trusses, Support Conditions, Types of Loadings, Classification of trusses, Determinacy of trusses, Basic assumptions of truss analysis, Method of joints, Method of sections.

Virtual work: Principle of Virtual Work, Active forces and active force diagram, Stability of equilibrium.

Faculty: AKHILESH PALIWAL (Assistant Professor)

SUB-MECHANISCS

of body at rest or in motion, when subjected to external mechanical disturbance.



Kinematics

<u>Statics:</u> Deals with forces in terms of their distribution and effect on a body in equelibrium i.e. at rest: <u>Dynamics:</u> Deals with the tody study of body in motion. <u>Kinematics:</u> is concerned with the description of motion of objects in dependent of causes of motion (distinuent, base) <u>Kinetics:</u> Both the motion and its causes are considered. <u>Matter:</u> Matter is anything that occupies space, <u>Matter:</u> Matter is anything that occupies space, <u>porseesses mass and offers resistance to any</u> <u>porticle:</u> A particle is an object that has infinitely <u>small</u> volume (occupies) ngligible space) but has a <u>mass</u> which can be considered to be concentrated at a point:

Body:- A body in which the distance b/w two particles remains constant j.e. the size and shafe of the body do not change, is called rigid body. Space:- is a region which extends in all directions and contains everything in iteg. Sun, moon, stors etc. Time:- is a measure of succession of events. The unit of time is second.

kinetics.

Motion: - When a body changes its position with respect to other bodies, then body is said to be in motion. It involves both space and time. Trajectory: - is a path followed by a body during its motion. It may be a straight line or a curve. Newton's law of motions:-First law:- Every body continues in it rest or of uniform motion in a straight line if there is no unbalanced force aeted upon it. Law of inortia. Second low: The rate of change of linear momentum directly proportional to the force and it takes plate in the direction of the impressed force. Third law: - To every action, there is equal and opposite reaction. Newton's law of gravitation: The mutual aftraction between two isolated bodies. i.e. Every body in the universe attracts every other body with a force directly proportion to the product of their masses and inversally proportional to the square of the distance separating $F \propto \frac{m,m}{r^2}$ $\Rightarrow \left[F = \frac{m_1 m_2}{r^2} \right]$ 61 = Universal gravitation constant = 6.67×10 m/19. se 2

Scalar:- A quantity is said to be scalar if it is completely. defined by its magnitude alone mass, longth, volume etc Vector: - Any quantity which possesses magnitude as well as direction is called a vector quantity. A vector quantity needs both magnitude and direction for its completé specéfication. i.e. désplacement, velocity, accrete. <u>Massi-</u> Mass is an indication of the quantity of matter present within a system. (kg) + unit Force:- force is an external agent which tends to change the speed or direction of a system. F=ma units $\rightarrow kg \times \frac{m}{S^2} = kg \frac{m}{S^2} = \lambda I$ Weight :- It is a force which the system exerts due to grovicitational acceleration. The weight (W) of a system equals the product of mass (m) and local gravitational acceleration (g). > mass of a system remains constant. > weight varies with change in value of gravetational acceleration from one place to another place. → generally the value of 19=9.81 m/s2] ensile force: - A member is saved to be in tension when it is subjected to two equal and opposite pulls and the members tends to elongate/increase in length. $\rightarrow P$ compressive force: - If a member is subjected to two equer and opposite pushes, and the member tends to shorten/decreas in length, the member is saved to be in compression. K-P + р —

System at forces:- when several forces of deifiteren magnitude and directions act upon a body, they constitute a force system. System of forces J. Liver Co-plange forces Non-coplanar forces. Non- concurrent concurrent Non-concurrent concurrent Collinear forces: The line of action of A forces Lie along the same straeght line. p <u>Ex:</u> Forces on a rope in a tug of war. co-planar parallel forces: The Lines of action of all forces are paraflel to each other and lie in a single plane Ex:- System of vortical loads acting on a bean loads acting on a beam. coplanar concurrent forces: All forces lie in the same plane have different directions but their lines of action act at one point. -> The point where the lines of action of the forces meet is known as the point of concurrency of the force system. Ex: forces on a rod resting against a wall. point of Pi concurrency P2

Diplanar non-concurrent forces: All forces lie in the same 3 > plane but their lines of oretion do not poss through a Single but 7/2 GX:- Porces on a laddor Single point. resting against wall and a two which is not at its control gravity. Non-coplanar concurrent forcer:- All forces do not lie in the some plane but their lines of action poes through a single point. PI Ex:- forces on a tripod corryin a camera. Non-coplanar and non-concurrent forces: - All forces do not Lie in the same plane and their lines of action do not meet at a single point. TPS 3P, on a moving bus.

Equilibrium: - When two or more than two forces act on a body in such a way that the body remains in a state of rest or of uniform motion. Then the system of forces is said to be in equilibrium.

Resultant: - withom a body is acted upon by a system of forces, then the vectorial sum of all the forces is know -> Resultant refers to the single force which produces the some effect as is done by the combined effect of several Equilibrant: - A number of forces may act on a body in such a manner that the body is not in equelibrium the resultant af several forces may cause a change of stat of rest or of uniform motion. A single force may have to be applied to the body to bring it in equilibriums. That single force is known as equilibrant. > Equilabrant is equal and opposite to the resultant of several forces acting on the body. Principle of Transmissibility: - When the point of application of a force acting on a body is should to any other point on the line of action of the forces without changing its direction, there occurs no change in the equilibrium state of the body i.e. This implies that a force acting at any point on a body may also be considered to act at any other point along its line of action without changing its effect on the body. (p) BX P. (A JP

 $P_1 = P_2 = P$

consider a force poeting at point A on rigid body. Bis another point on the line of action of force p. At point B, apply two oppositely directed forces (P, and P2) equal to and collinear with P. Sweh an application will in no way alter the action of given force p. At point A, forces Poind P, are equals but opposite and accordingly cancel each other. The leaves a force. = Pat B. This implies that a force acting at any point on a body may also be considered to act at any other point along its

The body in multiple concurrent forces aret on a body to krep The body in equilibrium, then resultant of these forces well be the vector sum of all these forces. >> The process of determining of resultant is known as composition of forces. Parallelogram law of forces: - The parallelogram law of forces is used to determine the resultant of two forces acting at a point in a plane and inclined to each other at an - angle. STATE MENT: - If two forces acting at a point be represented in magnetude and direction by the two adjacens sides of parallelogram, then their resultant is represented in magnitude end direction by the diagonal of the parallelogram passing through that point o R or Absind 0 O A good consider two forces pond & acting on a body at 'O'. The force p is represented in magnitude and direction by offushereas the force & is represented in magnitude and direction by JB. Let the angle b/w the two force be B. The resultant of these two forces is obtained by the diagonal oc of Drop I from c and let it meet of extend at parallelogram OAEB. the point D. In DCAD, side CA is parallel and equel to OB i.e. it represents force Q. From LOCD. OA = PAD=01080 $R = 0e = \sqrt{(0\lambda)^2 + (C\lambda)^2} = \sqrt{(0A + A\lambda)^2 + (C\lambda)^2}$ CDSSIMP $= R = \sqrt{(P + Q \cos \theta)^{2} + (Q \sin \theta)^{2}} = \sqrt{P^{2} + Q^{2} \cos^{2}\theta + 2PQ \cos^{2}\theta + Q^{2} \sin^{2}\theta}$ => R = JP2+ p2(sos20+sin20) + 2PQ cosQ $=) R = \sqrt{P^2 + Q^2 + 2PQ \cos Q}$

$$\begin{aligned} & + \operatorname{den} \alpha = \frac{C_{P}}{P + Q(\operatorname{ord})} \\ \Rightarrow & + \operatorname{den} \alpha = \frac{Q \sin \theta}{P + Q(\operatorname{ord})} \\ \Rightarrow & \alpha = + \operatorname{den}^{-1} \frac{Q \sin \theta}{P + Q(\operatorname{ord})} \end{aligned}$$

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De when the two forces act at night angle i.e. 0=90. $R = \sqrt{p^{2} + \sigma^{2} + 2pg \cos \theta} = \sqrt{p^{2} + \sigma^{2} + 2pg \cos g}$ $R = \sqrt{p^{2} + \sigma^{2} + 2pg \cos \theta} = \sqrt{p^{2} + \sigma^{2} + 2pg \cos g}$ $=)\left[\mathcal{R}=\sqrt{P^{2}\neq g^{2}}\right]$ $tamq = \frac{qsinq}{p+qcosq} = \frac{qsingo}{p+qcosqo} = \frac{q}{p}$ =) $\left| q = tan^{-1} \left(\frac{\partial}{\rho} \right) \right|$ (II) When the two forces act in the same line and some sense. i.e. D=D' app $R = \sqrt{p^2 + q^2 + 2pg \cos \theta} = \sqrt{p^2 + q^2 + 2pg \cos \theta}$ $= R = \sqrt{P^2 + Q^2 + 2Pg} = \sqrt{(P+g)^2}$ => (R = (P+g) ~ toox the resultant is maximum when For the forces are collinear and act in the same direction. IN When the two forces have the same line of action but opposite sense i.e. 0 = 180 (05180 = -1) $R = \int P^{2} + g^{2} + 2/Q \cos Q = \int P^{2} + g^{2} + 2PQ \cos BO$ $=> R = \sqrt{p^2 + q^2 - 2pq} = \sqrt{(p-q)^2}$ e ->P $\Rightarrow R = (P - q)$ The resultant is minimum when the two forces gre collinear but act in opposite direction.

I Two locomotives on opposite banks of a canal pull a vess moving parallel to the bonks by means of two horizontal sopes. The tensions in these ropes have been measured be 20 KM and 24 KM while the angle b/10 them is 60. Finan the resultant pull on the ressel and the angle blw each of the sope and the sides of the canal. 50/12 O B B-60 R C (cond) $R = \int P^2 + \theta^2 + 2P\theta \cos\theta = \int (24)^2 + (20)^2 + 2x2qx20x \cos^2\theta$ = \576+ 400+ 480 = \1456 = 38.16KN $tong = \frac{gsing}{P+gcosg} = \frac{20sin60}{24+20cos60} = \frac{20x0.866}{29+20x0.5} = 0.5094$ =) x = tem 0, 509 シ ベニ27° 止 $\beta = \theta - \alpha = 60 - 27 = 33^{\circ} E$ g. The magnitude of two forces is such that when actor at night angles produce a resultant force of Vio and when acting at 60° produce a resultent equal to vis . Hork out the magnitude of the two forces. $\frac{Sol''-}{2} R^2 = P^2 + q^2 + 2Pg \cos P - 0$ cosed when \$= 90 R² = p² + 0² + 2 pgcosgo => R² = p² + q² - 0000 $\Rightarrow (\sqrt{20})^2 = p^2 + p^2 \Rightarrow 20 = p^2 + p^2$ Coster when 0=60 R² = p²+ g²+2pg cos 60 => (V28)² = p²+g²+ 2pg k2 =) 28 = P2+ P2+PQ --(11)

From egn @ and D $20 = p^2 + p^2$ 28 = /p2 + /g2 + PB Start men ►+8 = + Pg = = PQ = 8 Squaring both sides we get $P^{2}q^{2} = 8^{2}$ $\Rightarrow Q^2 = \frac{64}{p^2}$ Put the value of g2 in eqn I we get $20 = p^2 + q^2$ $= = 20 = p^2 + \frac{64}{p^2}$ $\sum = 20P^2 = P^4 + 64$ $P^{4} = 20P^{2} + 64 = 0$ = p+ + + = - 16p2 + 64 = 0. :11 $\Rightarrow (p^2 - 16)(p^2 - 4) = 0$ $cohen (p^2 - 16) = 0$ 3 P= 16 =) P= 4 $cohm p^2 - 4 = 0$ $\Rightarrow P^2 = 4 \Rightarrow P = 2$ -'. P=2074. The corresponding value of force of will be 4 or 2.

I The resultont of two forces pand & adoug at a point is R The resultant & gets doubled when gis either doubled or direction is reversed. show that P. pand R conform to the norther $P: Q: R = \sqrt{2}: \sqrt{3}: \sqrt{2}$ 501 ... R2 = p2+ 92+2pg cosp ____ () when op is double R is also doubled (2R)2= P2+(2q)2+ 2P28cosp => 4 R2 = P2+ 4 P2+ 419 cosp. when direction of of is reversed the R is also doubled $(2R)^2 = p^2 + (-p)^2 + 2p(-p)\cos p$ =) 4R2 = p2+ 82-2pg cos8 -___(NT) Adding egn Dand (II) we get R²= P² + 9² + 2P8 % - 28 + 4R2 = P2+ 82 = 2p/gc050 5R2=(p2+92)+2 Eqn (1) x2 and adding in eqn (1) we get 8R2 = 2P2 + 292 - 4P8 for - Eque TX2 + 4R2 = p2 +49 2 + 4PB coso - Equi 12R2 = 3P2+492 => \$(4R2) = \$(P2+24P2) $\Rightarrow 4R^2 = P^2 / 2 P^2 - (v) \Rightarrow 2 P^2 = 4R^2 P^2 - (v)$ put the value of 2.92 in cgn I we get $= 5R^2 = 2P^2 + 4R^2 - P^2 = 5R^2 - 4R^2 = 2P^2 - P^2 = R^2 = P^2$ From eqn () we get $4P^2 = P^2 + 2Q^2 \Rightarrow 3P^2 = 2Q^2$ $\Rightarrow Q = \sqrt{3}_2 P^2 = \sqrt{3}_2 P$ $\Rightarrow P: Q: R = P: \sqrt{3}_2 P IP$ $\Rightarrow P: Q: R = \sqrt{2} \cdot \sqrt{3} \cdot \sqrt{2} Proved$ = R = P

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RESOLUTION OF FORCES: - Finding the component of a g force in twee directrions is called resolution of for resolution of forces. These compositions a correct mesone effect on the body as the same effect on the body as the given single force. > Let the given force R and Let it в be requiered to find its component o; 180 (x+B in directions making angle of and p with its line of action. Jat B coeith reference to other parallelo grom OACB the sides OA and OB represents the components in directions making ongle a of the given force R along ox and by respectively . i.P. OA= p and OB= Q LOCA = LBOE = B (By alternate ongles) LOAC = 180 - (X+B) Applying sine rule to DOAE, we get. AC = OB = R. AC = QC Sing sind sin[180-(x+B)] $\frac{P}{sing} = \frac{Q}{sing} = \frac{R}{sin} (\alpha + \beta)$ from egn D $\rho = \frac{R \sin \beta}{\sin (\alpha + \beta)}$ From eqn D $Q = \frac{R \sin \alpha}{\sin (\alpha + \beta)}$ ond => If ox and oy are at night engles OACD becomes à rectangle · · ×+B = 90' => B = 90- x

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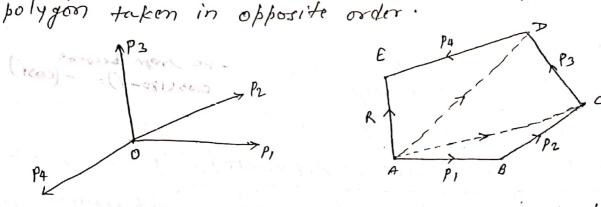
 $-\frac{P}{Sing(x+\beta)} = \frac{RSin(y_0-\alpha)}{Sing(x+\beta)} = \frac{RSin(y_0-\alpha)}{Sing(x+\beta)} = RCOSA$ 9 = <u>Rsing</u> = <u>Rsing</u> = <u>Rsing</u> THEOREM OF RESOLVED PARTS !statement: The algerbandic sum of the resolved parts of two forces in ageven direction is equal to the resolved part of their resultant in af the same direction. => Let de represents the of B' A' c' resultant of two forces out and ob acting at point o Further let ox be the direction along which forts or sesolved parts of OA, OB and or one to be worked out Drop perpendicular from B, A and C to meet of at B' A' and c' respectively. The triangles OBB' and AC> are congruent and that gives os' = AI = A'c' from the geometry, we haveoe' = oA' + A'c' = oA' + A' = oA' + OB'=> Resolved parts of force ore mox = Resolved part of of on ox+ Resolved part of force oBon ox.

8 RESULTANT OF COPLANAR-CONCURRENT FORCES !-Analytical method (Principle of resolved parts):-Offind the components of each force in the system in two 3 mutually perpendicular x and y directions. Make algebraic addition of components in each direction d3 to get two components a, 24 P4 € EFm and EFy. 1 Obtain the resultant both in magnitude and direction by combining the two component forces EFr and EFy which are mutically perpendicular. $\mathcal{R} = \sqrt{(\Sigma F_{21})^2 + (\Sigma F_{3})^2}$ and its inclination & to re-axis G. $t \operatorname{cm} \theta = \frac{\Sigma F y}{\overline{\Sigma} F x}$ Theorem of resolved parts: - STATEMENT: The algebraic sum of the resolved parts of two forces in a given direction is equal to the resolved ports of their resultant in the same direction From fig. P, P2, P3 and P4 are the concurrent 2 forces meeting at point 0 and making an angles a, , x2, x3 anda with ox. Tr Recolving along x-curis and y-axis, we get-EF2 = P, Cosd, + P2 Cosd2 + P3 Cosd3 + Pq Cosd4 EFy = P, sind, + P2 sind2 + P3 sind3 + P4 sind4 $R = \sqrt{(\Sigma F_{2})^{2} + (\Sigma F_{y})^{2}}$ EFY $fom \theta = \frac{\sum Fy}{\sum Fy} \left(\right)$

Equilibrium conditions af a particle:-A particle coill be in equilibrium when resultant of all the forces acting on it is zero. $R = \sqrt{(\Sigma F_n)^2 + (\Sigma F_y)^2} = 0$ $\Rightarrow (\Sigma F_{\mathcal{X}})^2 + (\Sigma F_{\mathcal{Y}})^2 = 0$ Now (EFu)² and (EFy)² are positive quantities and their sum com not be zero unless each of them is zero $\Sigma R_{1} = 0$ and $\Sigma F_{2} = 0$ Hence if any number of forces acting at a particle one in equilibrium, then the algebraic sum of their resolved parts in any two perpendicular directions are separately zero. TRIANGLE LAW OF FORCES .- STATEMENT: - gf two forces acting simultaneously on a body are represented by the sides of a triangle taken in order, their resultant is represented by the closing side of the triangle taken in the opposite order consider two forces acting or, the body as shown in fig. A B --- aV Q M Line AB be drawn to represent force pond BC to represent q. The triangle ABC is completed by drawin the closing line Ae. Line AC represents the resultant in magnitude, Line of action and direction. -> gt is to be noted that addition of force p and of in any order gives the same resultant R.

The following trigmometric relations can be applied while working out solutions by F2. the triangle law of forces. Y 180- 41 $\frac{F_1}{sin\alpha} = \frac{F_2}{sin\beta} = \frac{F_3}{sin\gamma}$ - vie rign because cos(180-V)= - (cost) $F_1^2 = F_2^2 + F_3^2 - 2F_2F_3\log r$ $F_2^2 = F_1^2 + F_3^2 = 2F_1F_3 \cos\beta$ $F_3^2 = F_1^2 + F_2^2 - 2F_1F_2 \cos\gamma$ Triongle lous of forces (STATEMENT):- it a system of three forces acting upon a body can be represented in magnitude and direction by the sides of a triangle take. in order, then the system of will be in equilibrium. 8 The three forces P. Q. and R acting at point O have been represented by the sides AB, BC and CA of the triang ABC. being equel and parallel to BC represents the Force Q. By the parallelogram/law of forces $\overrightarrow{AB} + \overrightarrow{AS} = \overrightarrow{AC}$ $\overrightarrow{P} + \overrightarrow{Q} = -\overrightarrow{CA} = -\overrightarrow{R}$:. Resultont of P. g. ond R = -R+R = 0 } Honce the system is in equilibrium.

PolyGION LAW OF FORCES: (STATEMENT): yet a number of concurrent forces acting simultaneously on a body are represented in magnitude and direction by the sides of a polygon, taken in order then the resultant is represented in magnitude and direction by the closing side of the



consider force Pr, P2, Bond P4 acting on the body at a point O. Line AB is drawn to represent force P1, Line BC to represent P2, Line C> to represent B and line DE to represent P4. The polygon is completed by drawing the closing line AE. This closing line AE represents the resultant of the given system in magnitude, and director

 $\overrightarrow{AC} = \overrightarrow{AB} + \overrightarrow{BC}$ $\overrightarrow{AD} = \overrightarrow{AC} + \overrightarrow{CD} = \overrightarrow{AB} + \overrightarrow{BC} + \overrightarrow{CD}$ $\overrightarrow{AE} = \overrightarrow{AD} + \overrightarrow{DC} = \overrightarrow{AB} + \overrightarrow{BC} + \overrightarrow{CD} + \overrightarrow{DC}$ $\overrightarrow{AE} = \overrightarrow{AD} + \overrightarrow{DC} = \overrightarrow{AB} + \overrightarrow{BC} + \overrightarrow{CD} + \overrightarrow{DC}$

$$\Rightarrow R = P_1 + P_2 + P_3 + P_4 \sqcup$$

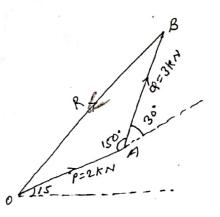
A stud is acted when by two forces
$$P_{A} = 2kN$$

and $Q = 3kN$ as shown in fig. Determine the
resultoment of these concurrent forces by using:-
D Parallelogram law of forces.
D Triangle law of forces.
 SDI^{n} :-
 $T = \sqrt{p^2 + Q^2 + 2pQcosQ}$
 $= \sqrt{2^2 + 3^2 + 2 \cdot 2 \cdot 3 \cdot Cos 30}$
 $= \sqrt{4 + 3 + 10 \cdot 33}$
 $= 4 \cdot 836$
 $t = \frac{Q \sin Q}{P + Q \cos Q} = \frac{3 \sin 3D}{2 + 3 \cos 30}$
 $= \frac{3x \cdot 5}{2 + 3 \cdot 2 \cdot 2} = 0.326$
 $\Rightarrow x = 18 \cdot 06^{\circ}$
 $= 18 \cdot 06 + 15^{\circ} = 33 \cdot 06^{\circ}$ \Box

$$\begin{array}{c} \textcircled{II} \\ \textcircled{II} \\ R^{2} = \rho^{2} + \varphi^{2} - 2\rho \varphi \cos 0AB \\ = 2^{2} + 3^{2} - 2 \times 3 \times \cos 15D \\ R^{2} = 4 + 9 + 1D \cdot 39 = 23 \cdot 39 \\ \Rightarrow R = \sqrt{23 \cdot 39} = 4 \cdot 836 \end{array}$$

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Applying the law of sines.

R = Q sin OAB = Sin AOB

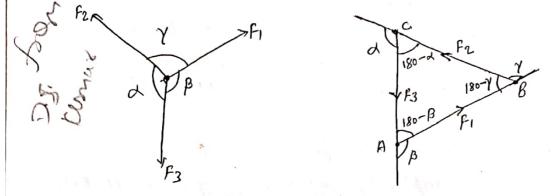
 $\Rightarrow Sin AOB = \frac{P}{R} Sin OAB$ = <u>3</u> x s in 150 = 0.310

=> LAOB = Sin-1 (0.310) = 18.06

- . Angle of inclination of the regultemt couth: the horizontal = 18.06 +15 = 33.06

At is essential to identify all external forces body of internal. " Exciting on a body of interest. "A correfully prepared sketch FREE BODY DIABIRAM (FBD):-That shows a body of interest seperated from all interacting bodies is known as free body deagram. It is simple diagram indicating the magnitude and direction of all forces acting upon the Object. A free body diagram has three essential Ogt is a diggram or skotch of the body. characteristics:-(The body is shown completely separated from all other The action on the body due to each body removed bodies. in the isolating process is shown as a force or forces in the diagram Example :-F·B·> (Ball) (Loudder, centre of gravity Hock

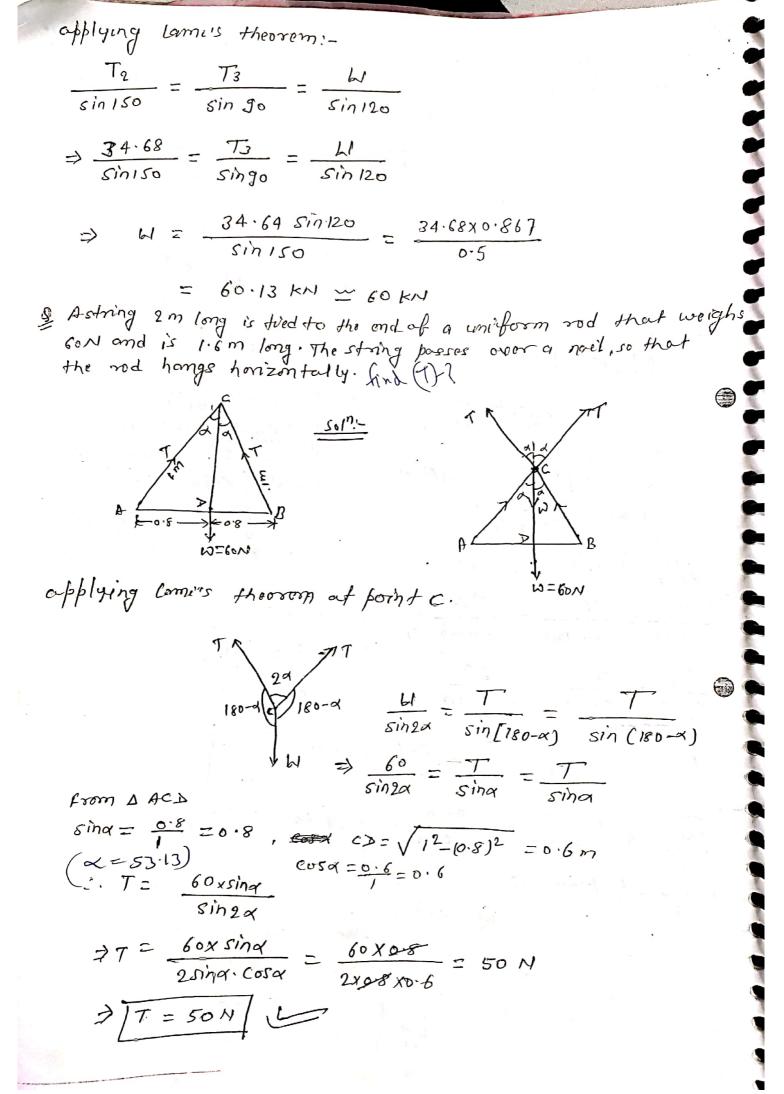
LAMI'S THEOREM'S STATEMENT - Statement - Sto a body is in equilibrium under the action of three the coplanar forces then each force is proportional to the sine of the angle b/w the other two forces.



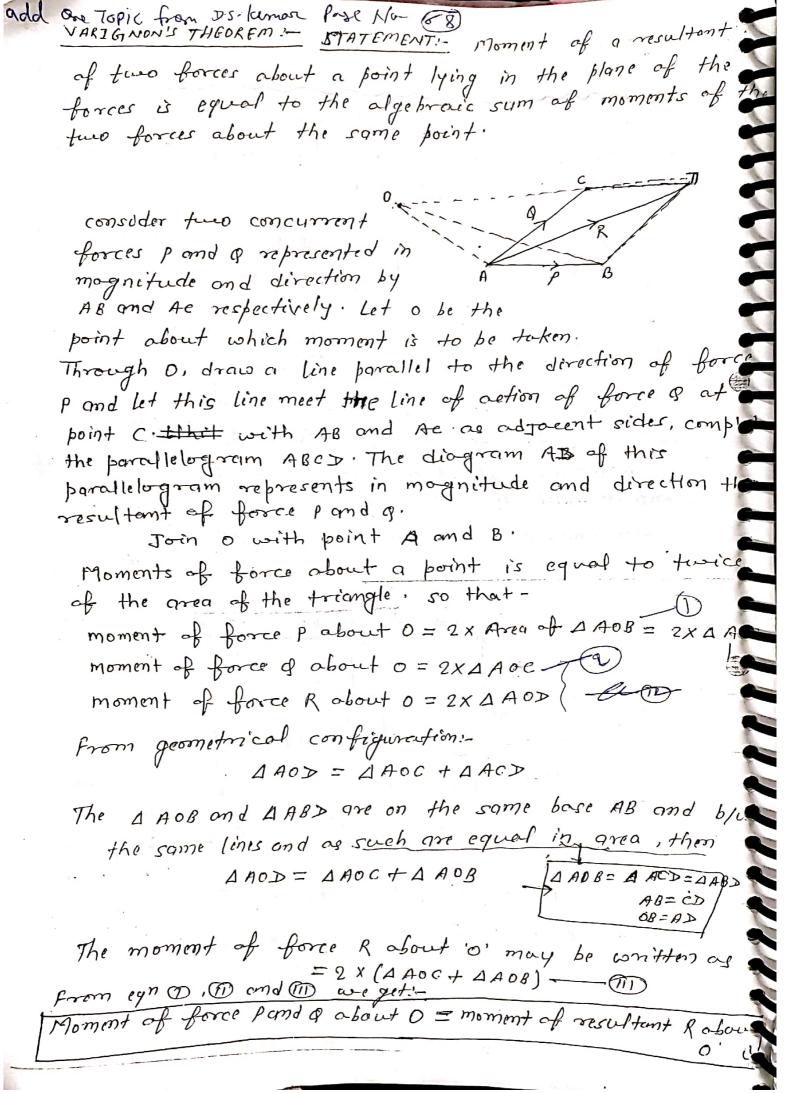
Draw the three forces F, F2 and F3 one after the other in direction and magnitude starting from boint A. since the body is in equilibrium (resultant is zero), the last point must be concide with A. Thus it results in a tring triangle of forces ABC. Now the external angle at A B and C are equal to B, Y and & respectively.

A cord supported at A and B carries a lead of golden at is
and a lead of H at C as shown in fig. Find the value of
it so that corremains harizonted:

$$\frac{dr}{dr} = \frac{1}{corr} = \frac{1}{cor$$



MOMENT: - The moment of a force about at point is defined is the tendency of the Borce to rotate a body about that apoint. > A moment has both magnitude and a direction, > Mathematically moment is defined as the product of the force and perpendicular distance. anticlockwise moment = - V? M = -P lclockwise moment the M=Pl PRINCIPLE OF MOMENTS:- The algebraic sum of Bll the clockwise moments about a point must be equal to sum of all anticlockwise moments - 2:0. clock wise moments = onticlockwise moments. RB RA If we take moments about point B. then EM =0 $R_A \times L - F_1 \times L_1 - F_2 \times L_2 - F_3 \times L_3 - F_4 \times L_4 + R_F \times L_5 = 0$ => RAXL + REXLS = FIXL, + F2XL2 + F3XL3 + F1XL4 clockwise moments enticlockweise moments.



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PARALLEL FORCES: - Forces which do not meet at a point but lie in the same plane are called coplanar paraflel forces. -> Like parallel forces (some direction) -> IJnlike parallel forces (All forces do not act in the same direction, COUPEE:- Two parallel forces equal in magnitude but opposite in direction and separated by a finite distance are saved to form a couple. -> Since sum of the forces of a couple in any direction is Zero. -> A couple has no tendency to translate a body in any 0 direction but tronds only to rotate the body on which 11it acts. -> The perpendicular distance separating the two forces is called arm of the couple. 1P - clockwise (twe) -> moment of a couple = (PX d) N-m -.l,-K-l2-_____ d -M=+PX-li + PX-la $=+P\left(J_{1}+J_{2}\right)$ M=+(PXd) NI-m LIORK :- When a force acts on a body and body under goes some displacement, then some work is done by force and it is equal to the F. K-d---+ K-d-> force multiplyed by distance FF - OC 1 × F travelled in the direction of force + if force is acting horizontally then work done W= FXd Nom/

+ if force & acting at a point at an angle & then workdome by the body [w = Frospid Mm]

conscept of VIRTUAL WORK: - 4& a body is in equilibrium under the applied forces, the workdone is zero. If we assume that the body in equilibrium, undergoes an infinitely small imaginary displacement (virtual displacement then some work will be imagined to be done. Such an imaginary work is called virtual work.

PRINCIPLE OF NIRTUAL WORK: - <u>STATEMENT</u>: If the virtual Loork done of particals or nigid bodies on a frictiontess System by all external forces and couples is zero all virtual displacements then the system is in equilibrium.

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ACTIVE FORCE: Active forces are external forces capable of doing virtual work during possible virtual displacement is force p and of are active forces because they would do work as the link move.

ACTEVE FORCE DIAGRAM." when using the method of virtual work, we draw a diagram that isolates the system under consideration. Unlike the free body diagram, where all forces are shown, the diagram for the method of virtual work needs to show only the active forces Since the reactive forces do not the the method of virtual work needs to show only the active forces Since the reactive forces do not the the method of wirtual work needs to show only the active forces do not the the method of wirtual work needs to show only the active forces do not the the method of wirtual work needs to show only the active forces do not the the the the the opplication of du = 0. Such a drawing will be termer as active force diagram. above figure is an active force diagram for the system shown.

Grophical Representation of Moment 2 Let fosce pribe Represented in magnitude & discition by vector AB. Let "O' be the Point about which moment is the be determined. P: Moment of fosce"p" about Point"". PXOM OM is I to AB ABXOM = 2x (2 ABXOM) Then = 2 (Area of DAOB). Statement: - The crost done and sigid body or a system of sisid budies in equilibrium, when a ctud upon A Set of forces, is zero for ony virtual displacement compatible aign constaninets on system. Upword force, forcer acting stowcools right and forces Sign Conventions in OC condiscution are considered (++e) D.F., L., GA.C.L. didudon an Fre)

An electric light fixture weighing 50N hongs from point C by two strings Ac and BC as shown in fig. Determine the forces in the strings Ac and BC. 50/":-45 30 T2 451.30 50N 150 180-45 = 185 50N $\frac{T_1}{Sings} = \frac{T_2}{sings} = \frac{50}{sings}$ $T_1 = \frac{50 \sin 135}{\sin 75} = 36-594 \text{ Å}$ $T_2 = 50 \sin 150 = 25.88 \text{ NI}$ Q. Two rollers of the same diameter are supported by an inclined plane and a vertical well as shown in fig. The upper and the lower rollers are respectively 200 N and 250N in weights. Assuming smooth surface Find the reactions induced at the points of support A, B, C and D. 0 A 5010-F.B.D at 02. >Rd (90+15) (180-15)

-

-

RRRRNN

~, = 200N

sin (30+15) singo

Rc

sin (180 - 15)

193.18 N Re = Rb RJ = 51.76N Ra 15. (-- 0) LOISITIS COSIS YRasin 15 15 Resolving the forces paralle to yw,cos15 the 0,02. R_{q} Cosis- w, sin is - $R_{d} = 0$ 10,=250 Rg = 120.61 N Resolving the forces I to the 0,02. Rb= Rasin 15+ 60, Cosis = 272.68 N. & Refer to the system of cylinders arranged as depicted in fig. The cylinders A and weigh 1000N cach and the weight of the cylinder Cis 2000 N. Defermine 1.2m the forces exprted at the contact points. ab=2- 0.6 + 0.6 = 1.4 m ac = 0.3+0.6 = 0.9m $\cos \alpha = \frac{1 \cdot 4/2}{\sigma \cdot 9} = \frac{\alpha d}{\alpha c} = \sigma \cdot 7777, \ \alpha = 38.94'$ applying Lamis theorem to the forces acting on sphere c. $\frac{R_1}{\sin(90+\alpha)} = \frac{K_2}{\sin(90+\alpha)} = \frac{2000}{\sin(180-2\alpha)}$ ⇒ R, = R2 = 1590.87 AL のみ() - (90 tà) consider the free body. diagram of cylinderA. 9000N EFx=0 => RA = RI COSOY = 1237.38 NIL EFy=0 $R_3 = \omega + R_1 sind$ TR3 = 1000 + 1590.87 Sin 38.94 = 1999.87 NIL

A simple supported beam at the ends, 5m spon carrier a load 15 kni at a distance of 2m from one end. Determine the af reaction using the principle of virtual work. Iskn пB 50/1:-ISKN laty Assume the rittual displacement given in resticut upper direction at point. B is y and point is ye then from the geometry $\frac{y_c}{y_c} = \frac{Ac}{AB}$ $\Rightarrow Y_{c} = \frac{AC}{AB} Y \Rightarrow \frac{2}{5} Y$ Here beam in equélibrieurs, Total workdone by these forces dup to virtual work must be zoro. $O \times R_A - Y_c \times IS + Y \times R_B = O$ => - == xyx+= + yx R8 = 0 => XX RB = & • 6 Y => [RB = GKN] The workdone by RA is zero because no desplacement. Now resolving forces resticutly we get $R_A + R_B = 1S = S R_A = 1S - R_B$ = RA = 15-6 = 9 KN/L

" Two beams All and CD of length 9 am and 10 am respectively are hingled at c. These are simply supported at A and B. Another supported is given at point B- The loads on the beam are shown in fig. Using the principle of virtua work find the reactions at the point B. 501"-1 6 m - [1] B C -7 m-K-6m-1 K-gm-+-10m-+ Rв Assume that a vertically displacement at the hinged po PH c in upward direction is y. From geomentary the displacement at point B and E may be calculated as. $\frac{y_{B}}{y} = \frac{AB}{Ae}$ \Rightarrow $y_B = \frac{AB}{AC} \times y \Rightarrow y_B = \frac{1}{9} y$ and $\frac{g_{e}}{g} = \frac{e}{c}$ $= \frac{1}{2} = \frac{$ As per principle of vertualwork the algebraic sum of the total workdome is zero. -: RANC+ YOXRB - JEX35+ ROXO = 0 =) JBXRB = YEX350 => = yx RB = = xx35p $= R_B = \frac{6x_35x_9}{7} = 270 \text{ N}$

Two beam AB and BF are supported on rollar at shown in fig. Determine its mand Determine the reaction at the rollar Brend & wing the method of virtual work. Rollor B E F RAZ 1_{Rc} RAY First of all consider the beam AB with rollar support at C. K 3m + 2m - K 1m lot y be virtual displacement in upward direction at point B 12 I's then from geometry we get EB $\frac{AD}{AD} = \frac{Y}{Y_c} = Y_c = \frac{AC}{AD} \times Y$ $z = \frac{1}{6} y$ The algebraic sum of the total virtual work must be - RAXO + ReX Jc- 1× Bo = 0 => Rexy= 1x Y = Rex = X = X $= R_{c} = \frac{6}{5} = 1.2 \text{ kN} \qquad \neq R_{c} = \frac{6}{5} \text{ kN}$ Now consider the beam BE, Let y be the virtual upward displacement at point B. 2Kn) E Pe-RFa 4 : from geometry RBK 2m K 3m K 3m K RFy $\frac{Y_c}{Y} = \frac{CF}{BF} \implies Y_c = \frac{CF}{RE} x y$ $= \frac{1}{2} \frac{$ Ye YE F $= \frac{3}{2} \quad y_c = \frac{3}{4} \quad y$

 $\frac{G}{Y} = \frac{EF}{BF}$ $\Rightarrow Y_F = \frac{EF}{BF} \times Y$ \Rightarrow $y_{E} = \frac{3}{3} y$ The appebraic sum of the total vistue work is zero Roxy - Roxyo - 2xXE + REXO = 0 = Roxy = Rox By + gx = y => Ryxy = (RcX + 3)y $= R_{B} = \frac{3}{4} \left(R_{c} + 1 \right)$ $= R_B = \frac{3}{4} \times (1.2+1)$ =) $R_B = \frac{3 \times 2.2}{4} = 1.65$

 vertical wall and a rough horizontal floor making an
 magle of 60° with the horizontal Detromine the book of
 finition in no friction at the floor using method at virtual work. Sol' .- Let, L-> Length of the lodder. Fa > Priction force at A. Let, virtual displacement on rough horizontal floor is du ν = l Cosθ virtual displacement. $dx = -l Sin \theta - d\theta$ Height of 'G' above base line Ac y= AGSinday = f Sind Virtual displacement at wall $dy = \frac{1}{2}\cos\theta \cdot d\theta$ There is also some virtual displacement at contre of gravity of uniform ladder. $dy = \frac{1}{2}\cos\theta \,d\theta$ According to the principle of virtual work Foxdu + Wxdy = O fade > workdone by friction Totalworkdone pil. Ridy > workdone by the wt of ladder. $\Rightarrow F_{a}(-lsin\theta d\theta) + \mu\left(\frac{1}{2}(os\theta d\theta) = 0\right)$ => Fa = 10 \$ coso do Ising to = Fa= W G+0 => for = 600 x 6+60' = 173.2 N=Fa /

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A block of wt. 2000N mests on a smooth inclined plane that makes on angle of 30° with the hmizontal. The block is supported by load P lying on another smooth plane of inclination 60 as shown in ge fig. The block and the load have been connected by om inelastic string . Determin the value of load P by using the method of virtual work sol"- small desplacement "on" of 2000N phrallel to Plane AB equals the displacement by of load p along the plane BC. 2 P. Con30 > When the load 2000N moves down the plane, the load p moves up the plane. -> The resolved parts of 2000N 2000N1 parallel and perpendecular to plane BC are "Psin 60" and "Pcos60" (0)60 -. Total virtual work = (2000 x singo') x dr - (Psin 60-x dy) -ve sign because of morrespend of the two loads are in opposite directions. .: From the principle of virtual work 2000 sin 30 x dx - PSin 60 x dy = 0 => 2000 sin 30 x dp = Psin 60 x dp dredy $P = 2000 \times \frac{5in30}{5in60} = 2000 \times \frac{0.5}{0.866}$ 7 P= 1154.73 N

A with ab loken is raised by two pulley system as sochown in big Determine the force F requered to hold the weight in equelebrius Assume that & goes down through a distance y. From the geometry of fig- it may be easily seen that weight moves upward by 1/24 distance. Long the principled of virtual work Fxy- wxy=0 =) Fxy= wxy => F = $\frac{10}{2} = \frac{10}{2} = 5kN$ Ablock of W,= 6 KN rests on the smooth surface inclined at 0=30 with the horizontal. The block is supported by an Desight We hung from a pulley as shown in fig. Using the Principal of virtual work determine the required we for Sequilibrium condition. Sol":- Assume that a virtual displacement y is given to block w, in the direction of inclined plane. From fig. it may be cusily seen that desplacement of who his is 1/2 in upwar direction. The displacement of block 30° (W, Singe) Din vertical directions is ysind in Jownward direction. By using the princepal of virtual work. W2× y - w, x ysing = 0 = w2X# = wixysind 00 7 w2 = 2 w, sing = 2x 6x sin 30 = x 6x1 = 6 KN1

2 A tiexagonal brame ABCDE is made up of six bars of equal. length and weight. One of the bar is fixed on a horizontal plane and the system lies in the vertical plane as shown in fig. The mid point of the two upper non-honizontal bens and connected by a string PR. Using the principal of virtual work show the tension in the string is "6 LI coto" where w' is the wit. of each bor and Q is the inclination of the inclined bars with the hon'zontal. sol"- Let "2a" be the length of each bar, Longth of string (PQ) = PG+GH+HQ F w = q coso + 2a + a coso =29+22050 differenting $dp_{\theta} = -20 \sin \theta \cdot d\theta$ The wot of each bar acts vortically downwards at its mod point. Let 21, y, z and to be the heights at the mid paints of AB, Bc and AF, chand FE, and ED as measured for the AB Then NZO dn = 0 dy = acosodo $y = asin \theta$ Z = 3asih Qdz = 39 Coso. do dh = 49 coso.do $h = 49sin \theta$ By the principal of virtual work (Wxdx)+ (wxdy+wxdy)+(wxdz+wxdz)+(wxdh)+(Txdpg)=0 bar BC bar AF bar CD bar FE => wx0 +(2wx a cos 0. d0)+(2wx 3 a cos 0. d0)+(wx 4 a cos 0 d0)+ Tx (-2a sin 0. d0)=0 => 2 wa corodo + 6 wa coso do + 4 wa coso do = Tx2a sin o do > 12 wd Coso do = T x 2/4 sind 40 6W Cost => / T = 660 Co+0 / 1

E A Ladder of 7m long 250N weight rests against a vertical wall with which it makes an angle of 45°, the co-efficient of friction blue the ladder of friction b/w the ladder and the wall is 0.4 and that b/w ladder and the floor o.s. Sif a man whose weight is one half of that of the ladder ascends it, betermine at what position we'll be induce sliping. 501 -Rp12 ·5-37 Givm:-AB = 7 m $\angle ABC = 45^{\circ}, \angle BAC = 45^{\circ}$ weight of laddor (WL) = 250N I leight of man (Um) = 125N 125N Co-efficient of friction b/w ladder and floor (W1) = 0.5 45 co-efficient of friction b/w C. ladder and wall (W2) = 0.4 'MI Let, n= Distance b/w a month and point A: Gits the mid point of the ladder at which the wf. 250N The man of wt 125 N is standing at point P which is at a distance » from A. Fi= MiRN, where, RN, > Normal reaction at A. F2 = 42 RN2 where RN2 -> Normal recretion of B. Resolving forces horizontally:- $R_{H_2} - f_1 = 0$ => RAZ= fi \Rightarrow $R_{N_2} = \mathcal{A}_i R_{N_i}$ $\Rightarrow R_{N_2} = 0.5 R_N,$

$$\frac{\operatorname{Rechning} \quad \operatorname{forces} \quad \operatorname{vertically}}{\operatorname{R_{N_1}} + \operatorname{fl}_2 - 250 - 125 \pm 0}$$

$$\Rightarrow \operatorname{R_{N_1}} + \operatorname{Il}_{\operatorname{R_{N_2}}} = 375$$

$$\Rightarrow \operatorname{R_{N_1}} + \operatorname{Il}_{\operatorname{R_{N_2}}} = 375$$

$$\Rightarrow \operatorname{R_{N_1}} + \operatorname{Il}_{\operatorname{R_{N_2}}} = 375$$

$$\operatorname{Fut} \quad \operatorname{He} \quad \operatorname{value} \quad \operatorname{of} \quad \operatorname{R_{N_2}} \quad \operatorname{form} \quad \operatorname{eq}^{n} \quad \operatorname{ID} \quad \operatorname{we} \quad \operatorname{get} \quad \operatorname{R_{N_1}} + 0.4 \left(\operatorname{os}_{\operatorname{R_{N_1}}} \right) = 375$$

$$\Rightarrow \operatorname{Il}_{\operatorname{N_1}} + 0.4 \left(\operatorname{os}_{\operatorname{R_{N_1}}} \right) = 375$$

$$\Rightarrow \operatorname{Il}_{\operatorname{R_{N_1}}} = 375$$

$$\Rightarrow \operatorname{Il}_{\operatorname{R_{N_1}}} = 375$$

$$\Rightarrow \operatorname{Il}_{\operatorname{R_{N_1}}} = 375$$

$$\Rightarrow \operatorname{R_{N_2}} = 0.5 \operatorname{R_{N_1}} = 0.5 \times 312.5 = 156.25 \operatorname{N}$$

$$\therefore \quad \operatorname{R_{N_2}} = 0.5 \operatorname{R_{N_1}} = 0.5 \times 312.5 = 156.25 \operatorname{N}$$

$$\therefore \quad \operatorname{R_{N_2}} = 0.5 \operatorname{R_{N_1}} = 0.5 \times 312.5 = 156.25 \operatorname{N}$$

$$\therefore \quad \operatorname{Il}_{\operatorname{R_2}} = -\operatorname{Il}_{\operatorname{R_{N_2}}} = 0.4 \times \operatorname{II}_{\operatorname{SG}} \operatorname{S}_{25} = 62.5 \operatorname{N}$$

$$\therefore \quad \operatorname{Il}_{\operatorname{R_2}} = -\operatorname{Il}_{\operatorname{R_{N_2}}} = 0.4 \times \operatorname{II}_{\operatorname{SG}} \operatorname{S}_{25} = 3.5 \times \operatorname{Il}_{2} = 2.47 \operatorname{m}$$

$$\operatorname{In} \quad \Delta \operatorname{AG_{H}} \qquad \operatorname{Cos} \operatorname{As}^* = \frac{\operatorname{AH}}{\operatorname{Ac_1}} \Rightarrow \operatorname{AH} = \operatorname{Ac_1} \operatorname{cs} \operatorname{As}^* = 3.5 \times \operatorname{Il}_{2} = 2.47 \operatorname{m}$$

$$\operatorname{In} \quad \Delta \operatorname{AG_{H}} \qquad \operatorname{Cos} \operatorname{As}^* = -\operatorname{Ae_1} \Rightarrow \operatorname{Ap} = \operatorname{Ap} \operatorname{cos} \operatorname{As}^* = 3.5 \times \operatorname{Il}_{2} = 2.47 \operatorname{m}$$

$$\operatorname{In} \quad \Delta \operatorname{AG_{H}} = 0$$

$$\operatorname{(125 \times \operatorname{AG}} + (25 \operatorname{ox} \operatorname{AH}) + \operatorname{(R_{e} \times \operatorname{Bc})} - (\operatorname{fe}_{2} \operatorname{Xac}) = 0$$

$$\operatorname{(125 \times \operatorname{AG}} + (25 \operatorname{ox} \operatorname{AH}) + \operatorname{(Il}_{5} \operatorname{cos} \operatorname{I}_{5} \operatorname{II}_{5} \operatorname{I}_{5} \operatorname{I}_{5}$$

Determine the minimum angle 0 at which a uniform ladder can be placed against a world without slepin under its own weight. The co-efficient of friction for all surfaces is 0.2. 50/:-Resolving forces along honizondally BTRB Ro = fo => Rs = -4 Ra fsind Resolving forces along vertically $R_{a} + f_{b} = W$ \Rightarrow $R_a + u/R_b = W$ $\Rightarrow R_0 + - Y(-YR_0) = W$ -1/2C050 - 42C050 + \Rightarrow $R_a + \Psi^2 R_a = h I$ lcord. $\Rightarrow R_{\rm G}\left(1+4/2\right) = 44$ \Rightarrow Rg = $\frac{L_1}{(1+cy^2)}$ Taking moments about end B. Rax Proso = WX & coso + fox lsind =) $\frac{4}{(1+y^2)} \cos \theta = \frac{4}{2} \cos \theta + \frac{4}{2} \cos \theta + \frac{4}{2} \cos \theta$ $= \frac{\omega l}{(1+u)^2} \cos \phi \pm \frac{\omega l}{2} \cos \phi + \frac{\omega l}{(1+u)^2} \sin \phi$ => $\frac{2 - (1 + u)^2}{2(1 + u)^2}$ cos $\theta = \frac{-u}{(1 + u)^2}$ sin θ =) $\frac{(1-42)}{2(1+42)}$ cose = $\frac{-4}{1+42}$ sine

 $= \frac{\frac{1 - \frac{1}{2}}{2(1 + \frac{1}{2})}}{\frac{1}{(1 + \frac{1}{2})}}$ => Sind Cord

= <u>1-42</u> x <u>(+42)</u> 2(1+42) x <u>-(++42)</u> => temp

> fin P = 1-42

87 (·

=) $fem \theta = \left[\frac{1-(0\cdot 2)^2}{2 \times 0\cdot 2}\right]$

=> D = tom = 67·38°